

Electromagnetic Multi Disc Toothed Clutches, Brakes & Clutch-Plates





ELECTROMAGNETIC MULTI DISC TOOTHED CLUTCHES, BRAKES & CLUTCH-PLATES

GENERALTECHNICALINFORMATION

Selection and sizing of clutches and brakes

Friction clutches and barkes can synchronize two pieces of machinery, which are rotating at different speeds; they absorb energy in the process. If safety requirements dictate the use of holding-current type brakes or clutches which must provide energy dissipation, the spring applied multiple-disc designs, which are suitable for wet or dry operation should be selected. Positioning functions in conjunctions with re-circulating ball mechanism can be performed by the zero-backlash spring-applied holding brakes-run dry, but not used as stopping single-disc clutches and brakes. These clutches and brakes are not included in the preview of this catalogue but can be had on request.

Sizing according to torque

M=9550. P/n in Nm

Where : P is power in kW n is the speed in min-1 Peak loads and shock must be taken into account.

Sizing according to energy dissipation, Operating frequency or life

It is necessary to distinguish between wet and dry operation.

Wet operation

For wet operation the wear of the friction surfaces is very slight if the sizing and lubrication are correct.

The energy dissipation can be calculated from the following equation :

$$W = J (m_2 - n_1)^2 \cdot M_s$$
 in Ws
182.4. (Ms ± Mi)

Where:

J in kg m² moment of inertia including inherent masses (in Sl units) n1 in min⁻¹ low speed n2 in min⁻¹ high speed Ms in Nm dynamic torque of clutch or brake (see table) M1 in Nm load torque Ms -M1 for acceleration Ms + M1 for deceleration W in Ws work

Dry operation

For dry operation a certain amount of wear takes place, depending on the energy to be dissipated, which limits the life of clutches. Incorrect sizing can cause increased wear rates and, in the case of multiple-disc clutches, distortion of the discs.



Permitted speeds with slipring-type clutches

Wet operation

The maximum permitted operating speeds of slipring-type clutches depend on the rubbing velocity permitted for the slipring. This may be up to 20 m/s when the slipring is carrying current; above 10m/s an additional dummy brush should be provided. A dummy brush is also necessary at lower rubbing velocities if the slipring is almost always carrying current. Rubbing velocities up to about 70 m/s are allowed when there is no current flowing.

Dry operation

Higher slip-ring rubbing velocities, and therefore higher rotational speeds, are permitted for dry operation, provided the current supply brushes are rigidly supported add there is no danger of contamination by oil or grease.

Permitted speeds with stationary-field clutches

In the case of stationary-field clutches the maximum permitted speed is determined by the maximum speed permitted for the bearing supporting the magnet body.

Higher speeds are permitted when the clutches have no such bearing (i.e. housing versions).

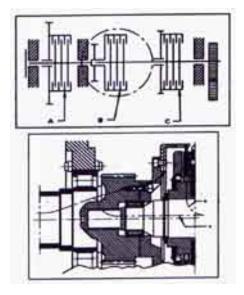
Vertical installation

Installation of multiple-disc clutches and brakes in the vertical position increases the drag torque in both wet and dry operation. The design must be such that the armature is at the bottom in order to prevent increased drag torque in the case of multiple disc clutches. Internal lubrication should be employed if possible for the wet operation for multiple-disc clutches in the vertical position. The disc clearance should be reduced to 0.2 mm; otherwise the engagement times will be greater.

Magnetic insulation

As well as magnetizing its own components, an electromagnetic clutch also magnetizes the shaft on which it is mounted and for this reason such clutches should not be mounted on machine spindles if at all possible.

With overhung clutches, it is possible to provide magnetic insulation in the form of bushes and carries of a non-magnetic material such as bronze or a magnetic bridge through which the magnetic flux can be transferred directly to the housing.



Without magnetic bridge a large amount of leakage flux passes through the carrier components and the clutch, adversely affecting the disengagement process and drag torque. Similar problems arise with a discontinuous gear shaft carrying several clutches. If clutch A or C is engaged and clutch B disengaged, for example, shunt induced in the shaft will flow through the carrier paths and discs of clutch B in the other shaft if there is no magnetic bridge.

The leakage flux in the disengaged clutch will cause increases drag torque, which can lead to overheating.

Whenever the physical arrangement of a clutch gives rise to the possibility of substantial magnetic flux or permanent flux in the working air gap of a disengaged or unexcited clutch, appropriate measures must be taken to prevent or divert the magnetic flow.

Lubrication and cooling for wet operation

Internal lubrication is recommended for multiple-disc clutches and brakes which provide energy dissipation, or are run at high speeds or are installed vertically. Ask for advice if necessary.

Splash lubrication or an external oil spray directed on to the disc pack is sufficient for other operating conditions.

In most cases the oil cloud provides adequate lubrication for the sliprings and for the magnet-body bearing of stationary-field clutches.

Oil quality

Only mineral oils with good resistance to aging and with neutral characteristics to copper and steel in the presence of slight condensation and elevated temperatures should be used for lubrication and cooling.

The oils must not have any electrolytic characteristics, which might promote oxidation or the formation of deposits, which could cause malfunction of the coil. In cases of doubt enquire form the oil supplier.





Oil viscosity

In accordance with DNL 43648 the catalogue values of torque and operating time refer to an oil viscosity, of 21 mm2/s. If the nominal ratings of the clutches are to be maintained the oil chosen must have a viscosity, at operating temperature, not exceeding that stated above. Thicker oils will reduce the torque and increase the operating times.

Oil flow rates

The oil flow rate for the internal lubrication of friction-type clutches and brakes should be approximately 0.1 to 0.2 1/min per clutch. An excessive flow rate will increase the engagement time and, when the energy dissipation is low, can lead to a reduction of the static torque. Electromagnetic clutches should not, if possible, run in oil because apart from affecting the engagement time and torque, the oil will be heated considerably by turbulence when running at high speeds. In the case of slipring-type clutches running in oil at high speed there is also danger of interruptions in the flow of current between slipring and brushes, which can damage the slipring.

The flows of oil for internal lubrication must be appropriate tp the power dissipation (i.e. the product of energy dissipation and operating frequency).

A proven empirical value is :

 $12 \text{ cm}_3 / \text{kJ}$

The oil flow through a clutch can be calculated from the following equation :

Q = W. z / 5000 in cm 3/minWhere

W = Energy dissipation per operation in Ws

z = Operating frequency in operations/hour

In most cases 0.1 and 0.2 I/min is adequate, especially for clutches of rated torque \leq 300 Nm. Excessive oil flow increases the drag torque, slows the engagement time and, especially at low speeds, reduces the rated torque. Insufficient oil leads to overheating, heavy wear and rapid destruction of the discs. When the flow is insufficient it is also possible for the oil to be heated so much that vapour is produced and the resulting oil/air mixture in the gearbox may be flammable. A spark in the gearbox (due to a poor electrical contact or rubbing between two parts, etc.) can give rise to an explosion which could possibility lift the cover of the gearbox.

Since this situation requires a specific stochiometric ratio of oil vapour to air and the simultaneous presence of an ignition spark, it is an extremely rare occurrence.

Nevertheless, it is advisable to take all appropriate measures to eliminate danger by observing the recommended temperatures and lubrication rates.

Moreover, gearboxes should have a large cover, which will lift at a low pressure and so avoid serious secondary damage.

Oil quantities in clutched gearboxes.

Experience has shown that main gearboxes whose clutches are providing high-energy dissipation require an oil capacity of approximately 5:1 per clutch or brake.

Feed gearboxes whose clutches provide less energy dissipation have been shown to require less oil and therefore less storage capacity.

Electrical circuits

The engagement and disengagement times of Electro-magnetically operated clutches and brakes are related to their size and design. Many years of experience have resulted in a range of clutches and brakes whose operating time and torque ratings satisfy all requirements over a wide range of applications.

The operating times indicated in the tables are based on rated voltage, normal operating temperature of the oil, and the use of the recommended varistors for surge protection. The operating times can also varied by mechanical methods. The normal circuitry is shown in Figs.

The clutch or brake coil is connected to the rated voltage via a contract or an electronic switching element. When the clutch is engaged (or brake applied) the rated voltage should be available at the coil connection itself. Allowance should be made for voltage drops due to long cable runs or high loads. Surge-protection devices must be used to avoid harmful voltages when the coil is switched off.



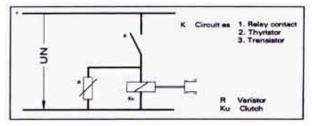
Surge-protection devices

The inductance of electromagnetic clutches causes a voltage peak when the supply current is switched off, which may exceed 1000V.

This may cause damage to the insulation or to the switching elements. A surge-protection device therefore has to be fitted to limit this voltage peak.

The following types of surge-protection devices can be used : 1. Varistor (non-linear resistors)

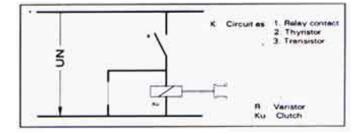
Recommended type with Order No. Q69-X3022



This type of varistor can be used with any operating voltage up to 30 V. The voltage peaks when the coil is switched off are limited to less than 100 V.

If the voltage of 30 V. exceeded on over excitation, a diode must be connected in series with the varistor, operating in the inverse voltage direction.

This type of circuit is suitable for all clutches and brakes shown in this catalogue. The voltage peak on switch off, approximately 90 V may still be excessive if sensitive contacts or solid-state circuits are used.



2. Zener diode

A diode must always be connected in series with the Zener diode

The Zener diode is particularly recommended for use with solid-state components because the induced voltage does not rise much above the Z voltage. With Z voltages in excess of 60 V the disengagement time corresponds to that stated in the catalogue. If the Z voltage is less than 60 V the disengagement time is increased.

3. Diode

A diode is primarily used for switching inductances with semiconductors. The use of a diode prevents a voltage peak when the coil is switched off. Allowance should be made for the fact that the disengagement time may be up to 5 times longer than that stated in the catalogue.

Slip-ring type toothed clutches

Construction and mode of operation

The slip ring-type toothed clutch can transmit a higher torque than a multiple-disc clutch of approximately the same size. It can be operated wet or dry.

A slip ring-type toothed clutch comprises a magnet body containing a potted coil, a toothed ring and slip-ring mounted on the outside and an armature disc. Springs on guide pins ensure that the two toothed rings are kept apart when the clutch is de-energized.

An adapter plate can be supplied for mounting the armature disc, although the customer can provide the necessary gear teeth on the mating part if he wishes.

When the coil is energized it produces magnetic flux.

The clutches operate from 24V DC supply.

The clutch can only be engaged when each tooth is opposite a tooth space.

The user must employ a suitable means of ensuring that the "tooth-to-tooth" position is avoided.



There must be no speed differential when the clutch is being engaged, except when there are torsionally flexible elements in the input and output, in which case a slight speed differential is permitted. The amount of differential can be determined by tests conducted on the item of machinery for which the clutch is intended.

Engagement shocks, which overload the toothed clutch causing it to disengage, must be avoided because the slipping of the teeth will ruin the clutch.

The value of static torque in the Selection tables must not be exceeded, however briefly.

The clutches can be disengaged under load and at any speed.

Unlike friction clutches, no figure of dynamic torque can be quoted for toothed clutches. They have only a static torque which is applicable when there is no relative movement between the teeth.

Such relative movement can be the result of shaft misalignment, sag or vibration.

The torque is transmitted without slip.

In contact to multiple-disc friction clutches there is no drag torque with a toothed clutch. Both toothed rings are made of nitrided steel and are therefore very hardwearing.

Stationary-field toothed clutches

The main applications for stationary-field toothed clutches are when access to the machine is poor and when high speeds make the use of slipring-type clutches difficult because of the restricted slipring rubbing velocity.

Construction and mode of operation

The stationary-field toothed clutch can transmit a higher torque than a stationary-field multiple-disc clutch of approximately the same size. It can be operated wet or dry, although wet operation is preferable.

A stationary-field toothed clutch comprises of a stationary magnet body containing a potted coil, bearings, a hub with a magnetic flux guide disc, toothed ring and an armature disc. Springs on guide pins ensure that the two toothed rings are kept apart when the clutch is deenergized.

An adapter plate can be supplied for mounting the armature disc, although the customer can provide the necessary gear teeth on the mating part if he wishes.

When the coil is energized it produces magnetic flux.

The clutches operate from 24V DC supply.

The clutch can only be engaged when each tooth is opposite a tooth space.

The user must employ a suitable means of ensuring that the "tooth-to-tooth" position is avoided.

There must be no speed differential when the clutch is being engaged, except when there are torsionally flexible elements in the input and output, in which case a slight speed differential is permitted. The amount of differential can be determined by tests conducted on the item of machinery for which the clutch in intended.

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The value of static torque given in the Selection tables must not be exceeded, however briefly. The clutches can be disengaged under load and at any speed.

Unlike friction clutches, no figure of dynamic torque can be quoted for toothed clutches. They have only a static torque which is applicable when there is no relative movement between the teeth. Such relative movement can be the result of shaft misalignment, sag or vibration. In contrast to multiple-disc friction clutches there is no drag torque with a toothed clutch. The torque is transmitted without slip. Both toothed rings are made of nitrided steel and are therefore very hardwearing.



Electromagnetic multi-disc friction clutches & brakes vis-à-vis toothed clutches :

A) Electromagnetic toothed clutches are used for very high killing speeds due to zero drag torque as compared to a slight residual friction torque in friction clutches.

B) Electromagnetic multi-disc clutches synchronize two pieces of machinery, which are rotating at completely different speeds, absorbing energy in the process. Electromagnetic toothed clutches can provide a dis-engageable connection between two shafts or pieces of machinery but can also be operated when shafts are stationary, running at same speed or at very slight differential speeds.

C) Size-to size, Electromagnetic toothed clutches can be suitably modified with respect to their tooth profile to get accurate positioning. E.g. : By suitably modifying the tooth profile, clutches can be designed to engage at a fixed point over the entire circumference of the clutch.

After studying the general technical information, we come to the range of clutches and brakes, which are presently covered in this catalogue.

VORTEX manufacture a very wide range of electromagnetic multi-disc and toothed clutches and brakes to cater to a wide plethora of application, primarily in the field of industrial machinery, drives and automation.

VORTEX range of electromagnetic clutches covered in this catalogue consist of two main types :

Friction type and Toothed type

1. Versions of magnetic lines of flux :

a) Flux-through plates : Friction combination is steel-steel and used for completely wet, oil immersed operations (BZ, CZ, AZ, PZ, SY, DY series) steel-steel and used for completely wet, oil immersed operations (BZ, CZ, AZ, PZ, SY, DY series)

b) Flux-outside plates : Friction combination is steel-sintered bronze for wet and intermittent wet. Dry versions use the steel-organic friction lining combination (KZ, LZ, FS)

2. Versions of magnetic field :

a) Rotating field : Slip-ring type clutches with rotating coil; current supply is through telescopic brush (BZ, CZ, AZ, PZ, EX, FX, KZ, LZ, DY series); These are available in single and double slip-ring versions.

b) Stationary field : Stationary coil versions with a driving rotor, bearings and fixed potted coil. (SX, SY series)

3. Functional versions :

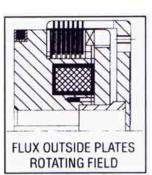
a)Clutch : Magnetic field causes the driving clutch-plate to transmit the drive to the driven clutch-plates (BZ, CZ, AZ, KZ, LZ, SY, DY series); These are available in single and double slip-ring versions.

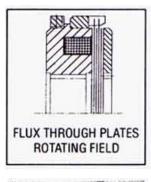
b)Brake : Magnetic field causes the driving clutch-plates to be braked by the stationary driven clutch-plates, which act as the brake plates (PZ, RZ series) Thus the slip-ring versions act as clutches while their equivalent non slip-ring versions are used as brakes.

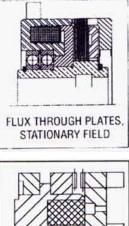
Version of mounting locations :

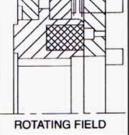
a) Mounting holes with central T-slot locations (EX, PZ, LZ series)

b) Mounting holes with bore location (FX, SX, BZ, AZ, LZ, DY, SZ, KZ series)













5. Versions of the operating environment :

a) Oil-immersed wet running : Clutches and brakes with the steel-steel friction combination are used with the oil functioning as the heat dissipation catalyst and simultaneously permitting larger friction pairs and hence the friction area, resulting in very compact design (BZ, CZ, AZ, DY, SY series)

b) Intermittent wet and dry running : Clutches and brakes with sintered-bronze / special material (paper, graphite) to result in very high torque and vastly improved wear-resistance properties, especially for severe heavy-duty applications (KZ, LZ, FS series)

c) Dry running : Clutches and brakes with steel-organic friction combination for dry, non-oil environment.

Note : Toothed clutches can be operated in either dry or wet environments, only precaution needs to be taken in the bearing versions by suitable greasing of the bearings (or use of enclosed greased bearings) for eg., in SX series

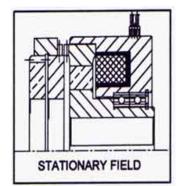
6. Armature versions (for toothed clutches)

a) Geared armature version for standard regular applications

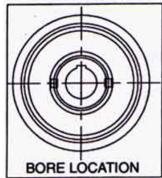
b) Slotted two-piece armature versions for applications demanding rapid disengagement with the maximum consistency.

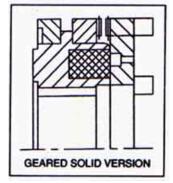
c) Backlash-free armature with leaf springs.

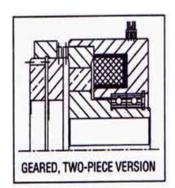
Toothed clutches have numerous versions based on the various possibilities with the toothed profiles to suite a very wide array of customer applications, hence fitting into a wide design possibilities due to their very high degree of flexibility.

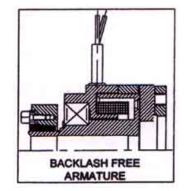












VERSIONS OF TOOTH PROFILE:

1. STANDARD

Torque transmission in both directions with minimum backlash.

2. STANDARD – BACKLASH FREE

Torque transmission in both directions of rotation without backlash.

3. OVERLOAD TEETH

Through increasing the flank angle, the torque capacity is reduced to approximately 50% of the normal torque. Transmits torque in both directions with little backlash. Only supplied with fixed position engagement.

4. SAW TOOTH - CLOCKWISE

Transmits the nominal torque in the clockwise direction. In the reverse direction approximately 10% of the torque can be transmitted. Engagement is possible at higher speeds.

5. SAW TOOTH – ANTICLOCKWISE

Transmits the nominal torque in the anti clockwise direction. In the reverse direction approximately 10% of the torque can be transmitted. Engagement is possible at higher speeds.

6. SPACED TEETH

Torque transmission in both directions with a large amount of backlash. Can be engaged at higher speed.

7. STEPPED TEETH - CLOCKWISE

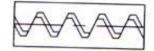
Transmits the torque in a clockwise direction with little backlash. In the apposite direction approximately 20% of the nominal torque is transmitted with a little backlash. Can be engaged at higher speeds. Only supplied with fixed position engagement.

8. STEPPED TEETH-ANTICLOCKWISE

Transmits the torque in anti-clockwise direction with little backlash. In the apposite direction approximately 20% of the nominal torque is transmitted with a little backlash. Can be engaged at higher speeds. Only supplied with fixed position engagement.

9. SELF LOCKING

Due to a very steep tooth flank angle, the teeth will not disengaged under load even with the power switched off. Torque transmission in both directions with little backlash.

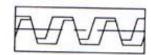


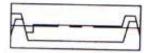


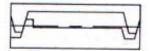












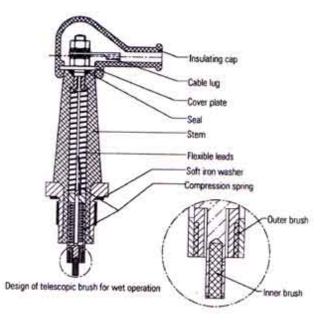






CURRENT SUPPLY BRUSHES FOR SLIPRING-TYPE CLUTCHES

Another very important component, especially with the slipring clutches are the telescopic current supply brushes.



Construction and mode of operation

Telescopic brushes for wet operation

The telescopic brush has been designed to give good transfer of current to the slipring despite the presence of a film of oil. The wire tips of the brush inserts are forced into contact with the case hardened slipring with high specific pressure and this penetrates the oil film.

The outer brush scrapes the oil from the splipring. The outer and inner brushes have different natural frequencies in order to ensure good contact in the presence of vibration because only one brush at a time can be in resonance. It also lengthens the life of the brushes.

Provided the telescopic brushes are fitted correctly it is possible for them to give trouble free services for years.

The figure shows a telescopic brush for wet operation. There are two brushes arranged concentrically and each is connected to the terminal by a separate flexible lead. Each brush has its own spring for providing the necessary down force. The steam is made of plastic and is moulded in one piece with the screwed body. An insulating cap covers the terminal.

Telescopic brushes are available in various different versions to suit requirements. Long brushes and dummy brushes are available in lengths between 30 and 100 mm (in 10 mm steps). These can be shortened by the customer to the size required, whenever necessary.

Brushes of normal length can be expected to have to be a longer life than the long versions because they have fewer tendencies to vibrate.

Worn brush inserts are easy to change; the amount of wear allowed is given in the operating instructions.

The less severe operating conditions with the smallest clutches enable a type of brush to be used having only one insert of two.

Slipring velocities of up to 20 m/s are allowed when carrying current. Above 10 m/s a dummy brush must also be fitted. It should also be fitted for velocities below 10m/s if the slipring is almost constantly carrying current. Dummy brushes are not connected to the electrical supply.



Dummy brushes for wet operation

Dummy brushes are similar in construction to telescopic brushes; the only difference is the omission of the cable lug at the top of the stem and the second hexagon nut and lock washer for securing the lug. It has a closed insulating cap.

VORTEX ENGINEERING WORKS

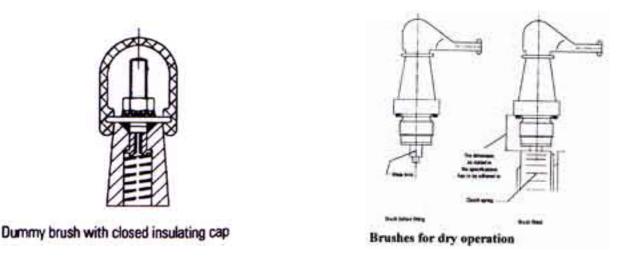


Fig. Shows a brush for wet operation prior to and after fitting. The same fitting constructions apply to both dummy brushes and brushes for dry operation

Brushes for dry operation

Current supply brushes for dry operation have a carbon insert. It is essential to keep oil and grease away from the slipring and brush if the clutch is to perform satisfactorily. Both the normal and long length versions of the brushes are available and there are different designs to suit all applications.

Dummy brushes are not required under dry operating conditions.

The design of a brush for dry operation is shown in the figure below. There is a central carbon brush connected to the terminal by a flexible lead. A spring produces the necessary down force. The stem is made of plate and is moulded in one piece with the screwed body. An insulating cap protects the terminal.

Worn brush inserts are easy to change; the amount of wear allowed is given in the operating instructions.

Earth return is used in the majority of applications because of the short life of a second slipring, or brush, for the negative pole would have at higher slipring velocities.

Contact us for more details and the wide varieties of current supply brushes we have to offer.



Design notes and installation

The specified brushes should always be used in order to guarantee good current transfer to the slipring.

1. Slipring velocities

Wet operation

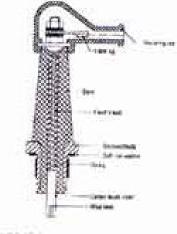
Slipring velocities upto 20 m/s are allowed when carrying current. Above 10 m/s a dummy brush must also be fitted.

A dummy brush should also be fitted for lower slipring velocities if the slipring is almost constantly carrying current.

Ordinary brushes and dummy brushes can only be distinguished externally by the different insulating caps. The dummy brush can be presented before or after the current supply brush.

Dummy brushes are not connected to the electrical supply.

Slipring velocities of up to 70 m/s are tolerable when de-energized; this may occur under some circumstances with gearboxes involving reverse driving.



Deeps Production By equation

Dry operation

Much higher slipring velocities are possible for dry operation provided the brushes and slipring are kept free from all traces of oil and grease. Dummy brushes are not needed for dry operating conditions.

2. Lubrication for wet operation

In gear boxes the oil splashes or clouds provides adequate lubrication for the brushes.

The slip-ring must not dip into oil.

A build-up of oil in front of the brush due to excessive lubrication can be harmful.

3. Installations

Proper installation with minimum vibration is essential for long life. If the brushes are not screwed directly into the machine housing they must be supported by strong bolted brackets on each side rigid enough to prevent any vibration.

4. Insertion Length

The specified insertion length "l" must be adhered to with a tolerance of - 1 mm, if the dimension is exceeded the down force of the brush will be insufficient and sparking may occur with subsequent pitting of the slipring.

Brushes of normal insertion length (14 and 22 mm) can be expected to have a longer life than the long types because they are less prone to vibration.

5. Polarity

The brushes must be connected to the positive pole of the supply. The life of the brushes and slipring will be reduced substantially if they are connected to the negative pole.

6. Fitting

The brushes should not be fitted in the oil sump because of the danger of short-circuiting due to metal particles in the oil. Also, unavoidable shrinkage of the plastic stem may result in leaks in the course of time.

7. Earthing

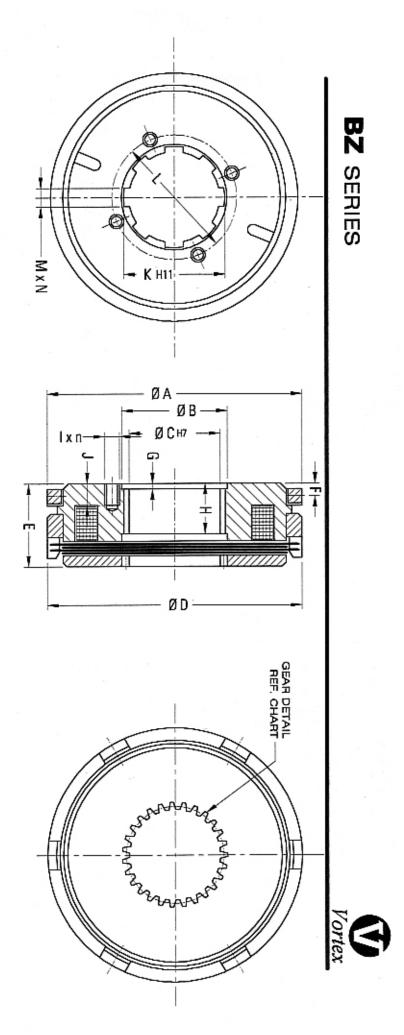
If a poor contact to earth (negative pole of the supply) is anticipated, e.g. when journal bearings are used, it will be necessary to provide a secondary conductive path giving a good connection between the clutch body and the machine housing or negative pole of the supply.

8. Special versions

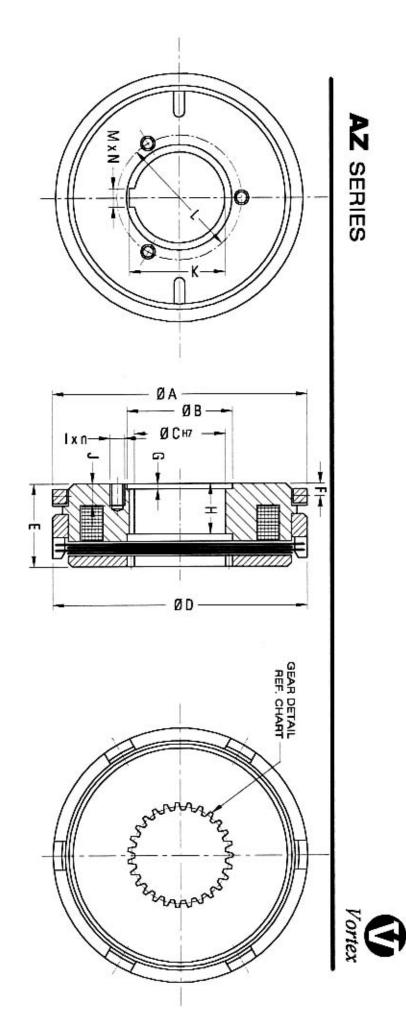
Different values of insertion length to those given in the tables can be obtained by shortening longer brushes.

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73.5	68.5	67	58.5	52	45.5	36	32	т	0
8.5	7	7	7	7	7	5.5	ธ.ธ	п	VERA
ω	ω	ω	2.5	N	2	1,5	1. ฮ	G	
35	33.5	31	30	26	23	20	18	I	MENS
MID	MIO	I,	MIO	MB	Me	MB	MB	I × I	OVERALL DIMENSIONS (MM)
σ	σ	1	σ	4	4	4	4	2	MM
20	18	1	15	15	12	10	10	Ċ	
96	88	78	78	58	50	6	36	~	
120	110	80	92	72	60	50	\$	-	
14	12	12	12	10	9	7	6	X X Z	
10	10	5	10	8	8	8	8	z	
96	33	25	27	28	27	27	22	teeth	GE
2.5	2.5	I	2.5	2.0	1.75 20°	1.5	1.5	Mod.	GEAR DATA
20°	20°	20°	20°	20°	20°	20°	20°	PIA	TA
Ø	6	ō	σ	5	5	4	4	-	2-
თ	6	6	σ	ŋ	ŋ	4	4	0	NO.OF
4.0	3.5	4. 0	N 01	2.0	1,8	1.2	1. 8	ĄG	
7.3	9.8	10	14	20	26	32	34	at 25° c (D ohms)	COIL
79	59	67	41	29	22	18	17	at 25° c (Watt)	
				24 V				(VOLTS)	POWER VOLTAGE

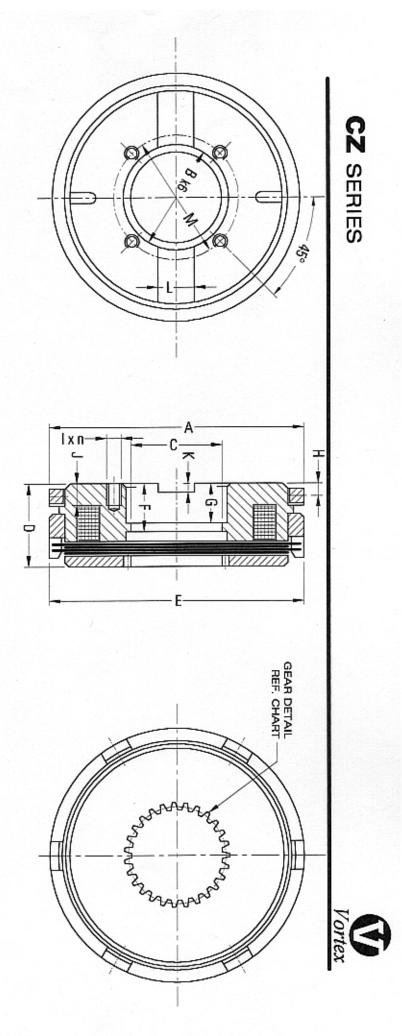


SR	MODEL	1	IN (kg-m)	3 8]。	VERAI		NENS	OVERALL DIMENSIONS (MM)	- <u>\$</u>		-		-		-	-	-		GEAR DATA	GEAR DATA	GEAR DATA PLATES	GEAR DATA PLATES
No	NO	STA.	-	DYN.	>	•	C	D	m	п	۵	I	Iхп	د	~	-		-	-	M × N	M X N No.of	M x N No.of Mod.	M X N No.of	M X N No.of Mod. P/A	M X N No.of Mod. P/A I O	M X N No.of Mod. P/A	M X N No.of Mod. P/A I O
-	0.5AZA	A 0.7	_	0.5	70	27	25	70	23	а.б	1.5	14	M4 3	0	26.5	ĊN	ω	32	32 6 1	6 1	6 1 15	6 1 15 1.5	6 1 15 1.5 20°	6 1 15 1.5	6 1 15 1.5 20° 2 2	6 1 15 1.5 20° 2	6 1 15 1.5 20° 2 2
N	01AZA		N	-	82	36	34	80	29.5	5.5	1.5	16.5	M4 3	10	35.7	1	4	41	41 6 1	f 9	6 1 20	6 1 20 1.5	6 1 20 1.5 20°	6 1 20 1.5 20° 4	6 1 20 1.5 20° 4 4	6 1 20 1.5 20° 4	6 1 20 1.5 20° 4 4
з	05AZA		10	ŋ	114	52	32	114	46	8	2	26	M8 4	12	34.2	4.4		60	60 10	60 10 1	60 10 1 27	60 10 1 27	60 10 1 27 1.75 20°	60 10 1 27 1.75 20° 5	60 10 1 27 1.75 20° 5 5	60 10 1 27 1.75 20° 5	60 10 1 27 1.75 20° 5 5
4	20AZA		\$	20	166	80	\$	166	58.5	7	2.5	эо	M10 5	15	45.4			92	92 16	92 16 2	92 16 2 27	92 16 2 27 2.5	92 16 2 27 2.5 20°	92 16 2 27 2.5	92 16 2 27 2.5 20° 5 5	92 16 2 27 2.5 20° 5	92 16 2 27 2.5 20° 5 5
σ	40AZA		70	20	195	90	₿	195	68.5	7	ы	33.5	M10 5	18	3 51.2		-	110	110 14	110 14 4	110 14 4 33	110 14 4 33 2.5	110 14 4 33 2.5 20°	110 14 4 33 2.5	110 14 4 33 2.5 20° 6 6	110 14 4 33 2.5 20° 6	110 14 4 33 2.5 20° 6 6
6	60AZB	-	110	60	210	100	09	210	73.5	7	ы	35	M10 5	20	56.2		-	120	120 14	120 14 4	120 14 4 36	120 14 4 36 2.5	120 14 4 36	120 14 4 36 2.5	120 14 4 36 2.5 20° 6 6	120 14 4 36 2.5 20° 6	120 14 4 36 2.5 20° 6 6
7	80AZA		180	100	240	110	75	240	77	8.5	ω	37	9 ZIW	20	79.1		-	136	136 20	136 20 4	136 20 4 42	136 20 4 42 2.5	136 20 4 42	136 20 4 42 2.5	136 20 4 42 2.5 20° 8 6	136 20 4 42 2.5 20° 6	136 20 4 42 2.5 20° 8 6
8	120AZA		240	130	258	123	68	258	80	8.5	з	39	M12 5	20	89.1	-	16	160 2	22	22 4	22 4 54	22 4 54 2.5	22 4 54 2.5 20°	22 4 54 2.5	22 4 54 2.5 20° 8 8	22 4 54 2.5 20° 8	22 4 54 2.5 20° 8 8



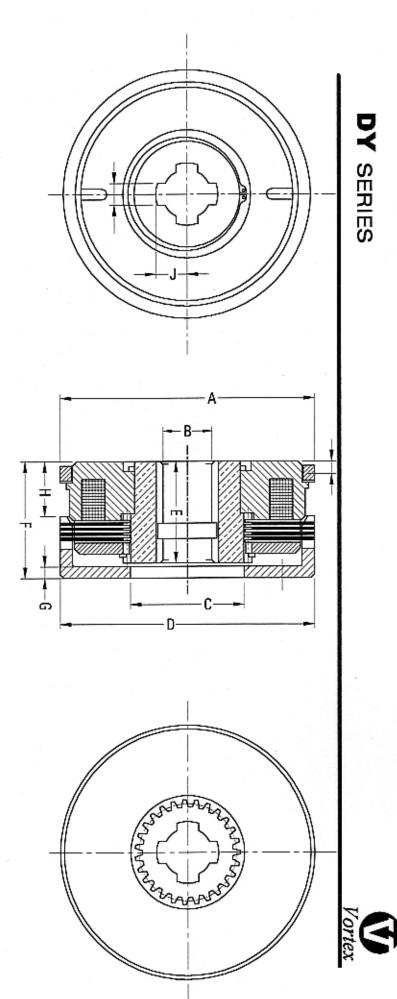
NOTE : AVAILABLE ON REQUEST, 160AZA & 320AZA.

_						_		124	
8	7	თ	đ	4	ω	N	-	NO.	SR
80CZA	60CZA	40CZB	20CZA	10CZA	05CZA	02CZA	01CZA	NO	MODEL
180	110	80	40	20	10	σ	N	STA.	IN (kg-m)
100	60	40	20	10	5.5	2	1	DYN.	TORQUE IN (kg-m)
240	210	195	166	140	114	95	82	A	
110	100	90	75	88	55	42	35	B	
100	06	80	66	09	45	37	31	0	
80	5,77	69	5.83	99	49.5	38	31	D	
240	210	195	166	140	114	93	80	т	
40	GΕ	34	OE	29	27	22	19	п	VERA
32	31	28	25	22	22	20	17	6	רר סו
14.5	14.5	13	13	11	11	7.5	7.5	п	MENS
M12	M12	M10	M10	MB	BW	M6	MB	-	OVERALL DIMENSIONS (MM)
4	4	4	4	4	4	4	4	хл	MM
18	16	12	10	8	7	σ	σ	C	
6	6	8	Ø	σ	σ	2.5	2.5	*	
25	20	20	20	16	14	12	12	-	
145	130	116	100	90	75	56	50	Σ	
42	36	εε	27	31	27	27	20	No.of	GE
2.5	2.5	2.5	N.5	2.0	1.75 20"	1.5	1.5	Mod.	GEAR DATA
20°	20°	20°	20°	20°	20"	20º	20°	P/A	TA
6	6	6	σ	ŋ	σ	4	4	-	₽z
6	6	6	σ	on N	σı	4	4	0	PLATES
5.0	4.0	3.5	2.5	2.0	1.8	1.2	i	AG G	
6.5	7.3	9,8	14	21	26	32	32	at 25° c (D ohms)	COIL
68	79	59	41	27	22	18	15	(Watt)	POWER
				24 <				(VOLTS)	POWER VOLTAGE



NOTE : AVAILABLE ON REQUEST, 120CZA,160CZA & 320CZA.

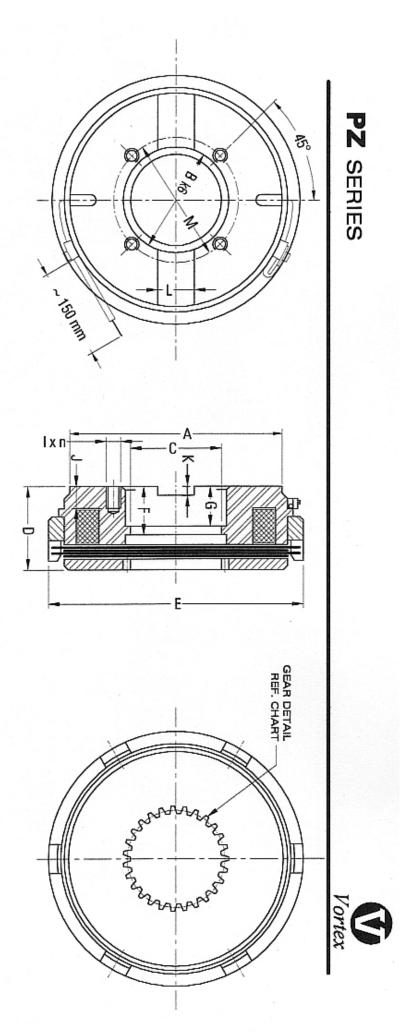
ſ	80	7	Ø	σ	4	ω	2	1	NO.	SR
	20DYB	20DYA	10DYD	10DYC	10DYB	05DYA	02DYA	01DYA	NO.	MODEL
	\$	40	20	20	20	10	σ	2	STA.	
Ī	20	20	10	10	10	5.5	N	L L	DYN.	Torque In (kg-m)
	166	166	134	134	134	114	95	82	⊳	
	42	46	42	30	40	28	21	16	œ	
NOTE BORE COULD BE SUITED TO CUSTOMER BEQUIREMENTS. ALSO AVAILABLE	75	76	60	60	70	51	45	34	ი	
	166	166	134	134	134	114	95	82	U	8
	64	64	56	56	56	49	41	33	m	ERALL
	71	71	61.5	61.5	61.5	55	46	38	п	OVERALL DIMENSIONS (MM)
	8	8	6	0	6	9	01	σ	G	IONS (N
	33	33	29	29	29	26	23,5	16	т	MM)
	12	14	12	8	12	10	8	a	×	
	1	4	1	1	-	4	4	1	ŝ	
	23.1	26	23.1	16.8	22.1	16	12.5	9.3	د	
	6	6	6	6	6	6	6	з	_	77
	5	ŋ	σ	σ	a	σι	4	ω	O AG	NO.OF
	2,5	N U	2.0	2.0	2.0	1.8	1.2	1.0	AG	υ'n
	14	14	20	20	20	26	32	39	at 25° c	RESISTANCE
	41	41	29	29	29	22	18	15	at 25° c (Watt)	POWER
					24 <				(VOLTS)	VOLTAGE



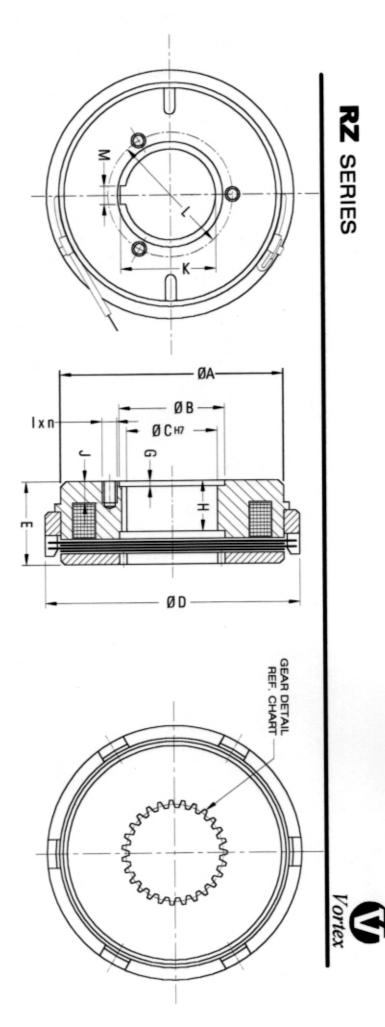
I on request, 40dya & 60dya. 5

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R	MODEL	N N	TORQUE IN (kg-m)	-					VERA		AENS	OVERALL DIMENSIONS (MM)	MM)					GEA	GEAR DATA	Ā	₽z	NO.OF	-	COIL	POWER	VOLTAGE
NO.	NO.	STA.	DYN.	Þ	œ	0	U	т	п	ດ	т	×	2	د	~	-	Z	No.of	Mod.	PIA	-	0	A G	at 25° c (Ω ohms)	at 25° o (Watt)	(VOLTS)
-	01PZA	N	-	87	36	31	31	80	19	17	7.5	МБ	4	σ	N 0	12	50	20	1.5	20°	4	4	1.0	39	15	
N	02PZA	σ	N	78	42	37	38	83	22	20	7.6	Me	4	σ	2.5	12	56	27	1.5	20°	4	4	is is	32	18	
ω	OSPZA	10	ភ ភ	85	66	45	49.5	114	27	22	11	М8	4	7	σ	14	76	27	1.75 20°	20°	σι	5 1	1.8	28	22	
4	1 OPZA	20	10	120	88	60	55	140	29	22	11	М8	4	۵	σ	16	99	31	2.0	20°	σι	01 N	2.0	21	27	22
σ	20PZA	40	20	142	76	66	58.5	166	30	26	51	M10	4	10	6	20	100	27	2.6	20°	5	01 N	2.5	14	41	
8	40PZA	80	40	170	90	80	69	185	3 4	28	13	M10	4	12	σ	20	116	33	N 0	20º	Ø	ອ ພ	а,5	9.8	59	
7	60PZA	110	80	184	100	90	77.6	210	39	31	14,5	M12	4	16	0	20	130	36	2.5	20º	8	0	4.0	7.3	79	
8	80PZA	180	100	216	110	100	80	240	40	32	14.5	M12	4	18	6	25	145	42	2.5	20	0	8	б.O	6.5	89	



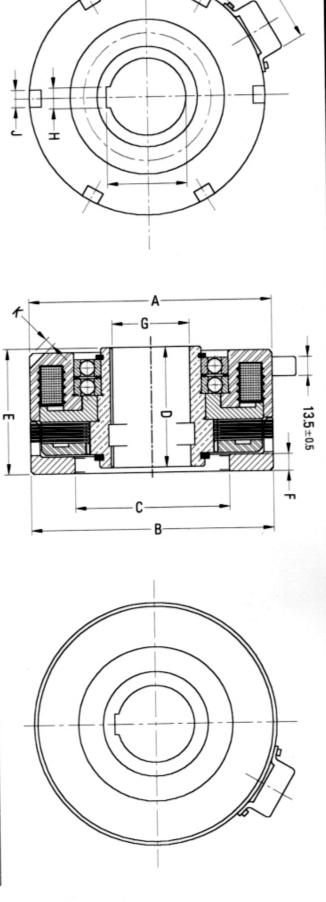
																		$\left \right $	$\left \right $	$\left \right $	ł	ł	ľ			
	130	4.6	4.0	8	8	200	2.5	54	2 4	22	1 160	89.1	20	M12 5	N 56	-	£ 08	258 8	85 25	-	6 123	216	130	240	120RZA	80
	88	6.5	5.0	6	6	200	2.5	42	4	6 20	1 136	79.1	20	M12 5	37 N		77 3	240 7	75 24	-	110	184	100	180	80RZA	٦
	79	7.3	4.0	6	0	200	2.5	36	4	0 14	2 120	56.2	20	M10 5	35 N	-	73.5 3	210 73	50 2	-	100	170	60	110	60RZB	6
	59	9.8	а.5	6	6	20°	2.5	33	4	0 14	2 110	51.2	18	M10 5	33.5 N	_	68.5 3	195 68	45 18		80	142	20	70	40RZA	σı
24 V	41	14	2.5	σ	o	20°	2.5	27	8	2 16	4 92	45.4	15	M10 5	N 05		58.5 2.5	166 58	40 16	4	80	120	20	40	20RZA	4
	22	26	1.8	σ	σ	200	1.75	27	-	10	2 60	34.2	12	M6 4	26 N		46 2	114 4	32 11	з	52	95	5	10	05RZA	з
	15	39	1.2	4	4	200	1.5	20	-	6	7 41	35.7	10	M4 3	16.5 N		29.5 1.5		4 80	34	36	78	-	N	01RZA	N
	10	55	0.7	N	N	20°	1.5	15	-	6	5 32	26.5	6	M4 3	14 N		23 1.5		5 70	25	27	67	0.5	0.7	0.5RZA	-
(VOLTS)	(Watt)	(D ohms)	₽G	0	-	PIA	Mod.	teeth	N X D	Σ	-	*	c	чхл	I		E G		0	0	B	>	DYN.	STA.	NO.	NO.
VOLTAGE		×		NO.OF	-	ATA	GEAR DATA	GE					Ξ	OVERALL DIMENSIONS (MM)	ENSIO	DIM	VERAL	0					TORQUE IN (kg-m)	IN (k	MODEL	SR



NOTE : AVAILABLE ON REQUEST, 160RZA & 320RZA.

NOTE :ALSO AVAILABLE WITH DIFFE
NOTE :ALSO AVAILABLE WITH DIFFERENT BORE DIMENSIONS AS PER CUSTOMER REQUIREMENTS. ALSO AVAILABLE WITHCONNECTING WIRE LEADS.
OMER REQUIREMENTS.

	G		3.0	σ	a	8	10	63	18	80	12	94	85	102	195	195	4	80	40SYA	20	
1	B	u d			ţ,	a		54	14	5	8	8	82	8	165	165	зо	50	30SYA	7	
1	000		2.0	, o			6	38	8	36		78	67	100	165	165	25	40	20SYA	8	
	8	0		1 0		a	0	52.5	14	50	•	8	67	ő	157	167	20	40	20SYB	σ	
	8	D d		n 0		σ		33.3	8	з	a	72	65	ĩ	147	147	15	25	15SYA	4	
1	7 0			n 0	, ,	σ		42.5	12	\$	8	8	8	8	134	134	10	20	10SYA	ω	
1	3	D -	3	1 0	, ,	4	a	31.9	8	зо	0	64.5	67	70	114	114	σ	10	05SYA	N	
	43	14	, i	4	, a	. u	0	28.3	8	25	σ	58	52	8	80	95	2.5	4	02SYA	-	
	(Well)	(Ω ohms)	A AG	• •	- 1	~	c	-	I	ရ	-	m	0	n	8	Þ	DYN.	STA.	NO.	NO.	
	at 25° c	RESISTANCE at 25° c	8	PLATES	_			1	Š	OVERALL DIMENSIONS (MM)	IMENSI	ERALL D	Q				TORQUE IN (kg-m)	IN (k	MODEL	SR	



FOL

18.5 ±1

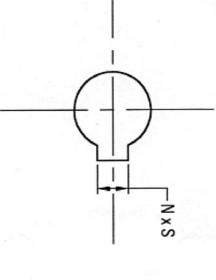
SY SERIES

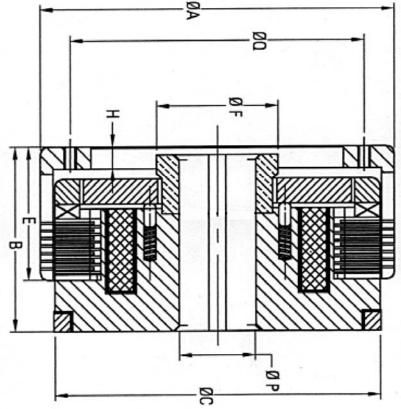
Vortex

4

NOTE : ALSO AVAILABLE : 250 KZA-S, 400 KZA-S, 630 KZA-S AND 1000 KZA-S ON REQUEST (STEEL - SINTERED BRONZE Combination) ALSO AVAILABLE : KZA-T VERSIONS OF ALL SIZES FOR DRY APPLICATIONS (STEEL - ORGANIC LINING Combination)

SR	MODEL	ТО	TORQUE	0	/ERAL	LDIN	OVERALL DIMENSIONS (MM)) SNG	MM)		MO	MOUNTING DIM. (mm)	DIM.	KEY WAY	KEY WAY DIM.(mm)	COIL	CONSUMP.	VOLTAGE
NO.	NO.	IN (k	IN (kg-Nm)	>	8	0	ш	п	G	т	Q	NO.OF	SIZE	(MAX)	(N x S)	at 25°C a (ohms)	at 25°C (WATTS)	(VOLIS
-	01KZA-S	20	12	100	45	100	33	35	42	σ	8	4	M6	22	2×6	22	26	
N	02KZA-S	40	25	110	48	110	34	42	45	5	90	4	M6	28	2×6	22	27	
ω	04KZA-S	63	40	120	52	120	39	48	48	6	100	6	M6	32	3×6	17	33	
4	06KZA-S	100	63	132	55	132	\$	52	50	7	105	6	M8	35	3×6	13	43	
σ	10KZA-S	160	100	147	83	145	45	58	53	7	120	6	M8	42	8 X E	13	43	
6	16KZA-S	250	160	162	62	160	47	65	57	7	135	6	8M	48	8×E	12	47	240
7	25KZA-S	400	250	182	68	180	52	72	63	8	155	6	M10	55	3 × 10	10.5	55	
8	41KZA-S	630	400	202	76	202	61	82	70	9	170	6	M10	60	3 × 10	9.3	62	
ω	64KZA-S	1000	630	235	86	230	70	95	80	10	200	6	M12	70	3 × 14	8.3	70	
ð	100KZA-S 1600 1000	1600	1000	270 100	100	255	75	105	92	12	235	6	M12	80	3 × 14	7.3	79	
=	11 161KZA-S 2500 1600 310 115	2500	1600	310	115	295	90	120	120 107	14	260	6	M12	90	3 x 16	9	97	



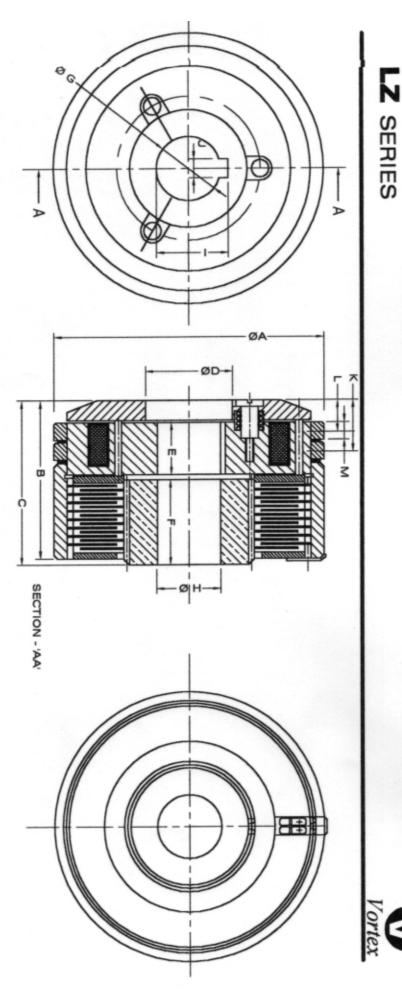


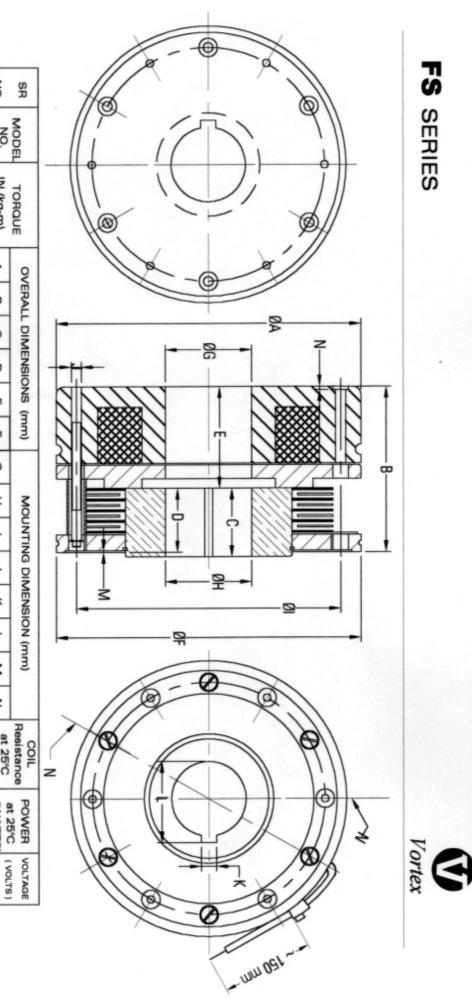
KZ SERIES

Vortex

NOTE : ALSO AVAILABLE IN STEEL - ORGANIC LINING Combination (LZA - T series) for dry applications ALSO AVAILABLE IN STEEL - SINTERED BRONZE Combination (LZA - S series) for wet applications

SR I	MODEL	TORQUE	Q	ERAL	OVERALL DIMENSIONS (MM)	ENSIO	NS ()	MM)	M	(mm)	(mm)	N.	SLIP	SLIPRING DIM. (mm)	DIM.	RESISTANCE	CONSUMP.	VOLTAG
NO.	NO.	IN (kg-m)	>		0	o	m	п	G	I	-	L	*	-	٤	AT 25°C Ω (ohms)	AT 25°C (watts)	(volts)
-1	03 LZA	2.5	95.5	43.2	46.2	43	21.5	14	36	36	37.5	10	20	10.7	7.6	29	20	
N	05 LZA	σ	116.5	54.5	57.5	67	23.5	21.5	48	48	50.9	14	22.5	12.7	8	26	22	
ω	10 LZA	10	140.5	64.5	67.5	66	24	28.5	56	56	59.4	16	24.3	14	8.5	17	31	
4	20 LZA	20	166.5	70.3	73.3	78	26	31	66	66	69.8	20	27.5	16.8	8.5	11	55	24V
a	40 LZA	40	210.5	80.5	83.5	83	31	33	80	80	1	1	33.2	20.6	10	8	76	
0	80 LZA	80	240.5	102	105	109	41	39	95	95	1	1	40.2	23	10	8	110	
4	160 LZA	160	295.5	110	113	125	41	44.5	110	110	1	1	43.5	25.5	12	4	160	

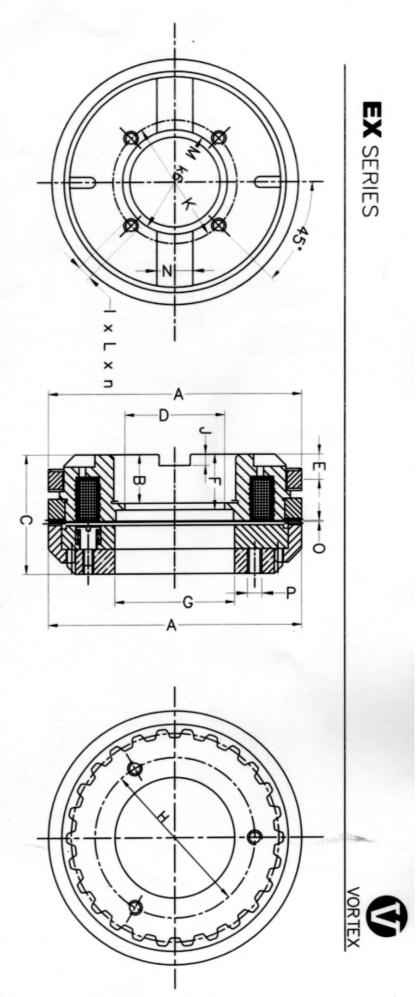




SR	MODEL	TORQUE	Q	OVERALL DIMENSIONS	DIM	ENSIC		(mm)		NO	UNTIN	IG DIN	MENS	MOUNTING DIMENSION (mm)	(mr		COIL Resistance		VOLTAGE
NO.	NO.	7	>	8	0	D	m	п	G	I	-	C	*	-	Σ	z	at 25°C (ohms)	(WATTS)	(VOLTS)
-	01 FSB	0.8	100	61	30	20	41	88	31	25	75	5.5	6	27.5	2.5	1.5	21	28	
2	02 FSB	1.8	115	65	35	23.5	41.5	100	39	35	90	5.5	8	38.3	2.5	1.5	15	39	
ω	03 FSB	3.5	135	75	40	27.0	48	120	45	40	110	6.5	8	43.3	2.5	5.1	13	43	24 v
4	08 FSB	7.5	165	95	45	34.5	60.5	150	62	50	140	6.5	12	53.3	2.5	2	11	54	
σ	15 FSB	15	190	105	55	37.5	67.5	170	67	60	160	8.5	14	63.8	з	ε	9	108	_
6	30 FSB	30	220	120	60	45	75	195	72	60	180	10.5	14	63.8	з	ε	9	124	
7	60 FSB	60	250	138	70	54	84	222	80	70	205	12.5	18	74.4	4.5	4	4	139	

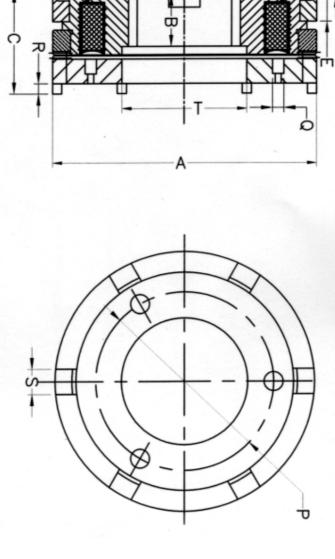
AVAILABLE
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EXA

70	MO	0./	22		ī	0	0	01 71W	2	2	Ca l	1.0	30	11	-	212	000	OULAN	-
		2	3	-	4 40	0			-	-	-	-	3	1		2		EDEV.	1
									┢	┝	┝								
7.4 77	M6	0.7	20	110	125	6	U)	M10 14	150	110 1	34	13	100	67	28	195	220	40EXA	6
						t										I			
8.3 69 21	M6	0.7	20	75	100	6	5	M10 9	120	80	30	13	65	60	25	166	120	20EXA	5
12 49 24 V	MS	0.7	16	68	90	5	4	M8 8	100	70 1	28	=	60	54	22	140	60	10EXA	4
14 42	M4	0.7	14	55	75	5	4	M8 6	80	53	25	=	45	47	22	114	35	05EXA	3
24 24	M4	0.7	12	42	56	2.5	4	M6 5	65	45	22	7.5	37	40	20	95	20	02EXA	2
34 17	M4	0.5	12	35	50	2.5	w	M5 5	55	35	22.5	7.5	31	39	20	82	10	OIEXA	-
at 25° c at 25° c (VOLTS) (Ω ohma) (Watt)	P	0	z	ĸ	*	c	× >	I X L X	т	G	۳	m	•	с	8	>	IN (kg-m)	NO.	NO.
					Š	M) S	NOISN	OVERALL DIMENSIONS (MM	OVER								TORQUE	MODEL	SR



NOTE : ALSO AVAILABLE MTH GEAR ADAPTER PLATES AND GEARED DRIVEN END (EXA SERIES) ALSO AVAILABLE 60EXB, 80EXB AND 120EXB ON REQUEST ALSO AVAILABLE SPECIAL BACKLASH FREE VERSIONS ON REQUEST

	77	7.4	0.7	15	15	15	150	MIO	125	6	110	8	10	67	110	28	110	195	220	40EXB	6
	69	8.3	0.7	12	12	12	120	MIO	100	6	92	8	10	60	80	25	75	166	120	20EXB	თ
	49	12	0.7	10	10	12	100	8M	90	5	80	6	10	54	70	22	68	140	60	10EXB	4
24 v	42	14	0.7	8	6	6	80	8M	75	5	60	3	10	47	53	22	55	114	35	05EXB	з
	24	24	0.7	8	6	9	65	M6	56	2.5	50	3	10	40	45	20	42	95	20	02EXB	2
	17	34	0.5	8	6	9	55	M5	50	2.5	41	3.5	8	39	35	20	35	82	10	OIEXB	-
(VOLTS)	at 25°C (WATTS)	at 25°C a (ohms)	(mm)	s	R	٥	σ	SIZE OF	-	×	د	G	٦	m	D	0	8	>	IN (kg-m)	NO.	NO.
VOLTAGE	POWER CONSUMP.	COIL	AIR))	DRIVE END	DRIV	DIME	DRIVE END DIMENSIONS(mm)	DRIVE END	DRI	₽		(MM)	I) Sh	INSIO	OVERALL DIMENSIONS	VERAL	0	TORQUE	MODEL	SR



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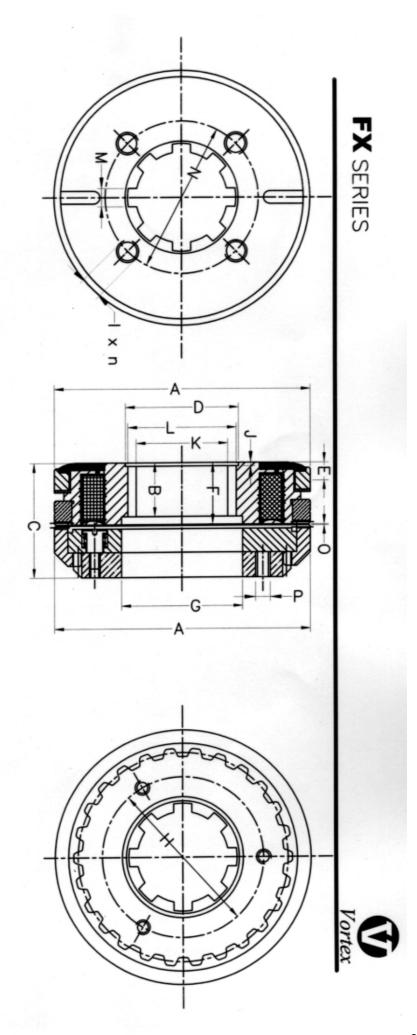
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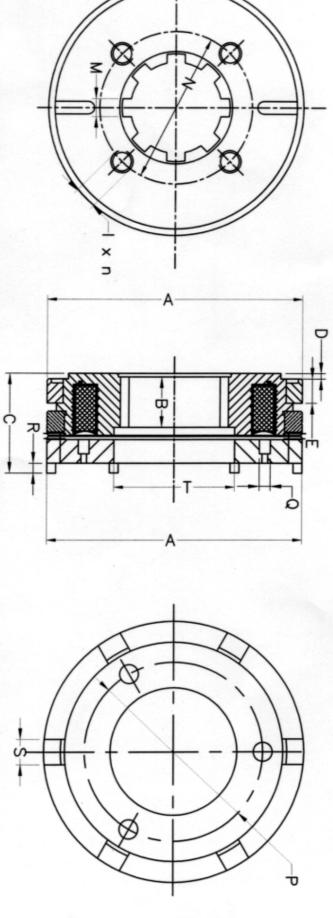
NOTE :
AVAILABLE
ş
REQUEST,
120FXA
&160FXA.

SR	MODEL	TORQUE								OVE	RALL	OVERALL DIMENSIONS	SNOISI	(MM	5						COIL	POWER	VOLTAGE
NO.	NO.	IN (kg-m)	>	8	0	0	m	٦	G	т	- ×	3	c	*	-	M × n	3	z	0	P	at 25° c (A ahms)	at 25° c (Watt)	(VOLTS)
-	OIFXA	10	82	23	37	36	5.5	23	35	55	M4	ω	1.5	34	36	6	ы	4	0.5	M4	34	17	
2	02FXA	20	95	20	38	42	5.5	23	45	65	M6	4	1.5	36	40	7	8	50	0.7	M4	24	24	
ω	OSFXA	35	114	23	43	52	6	26	53	80	M6	4	2	46	50	9	80	60	0.7	M4	14	42	
4	10FXA	60	140	26	51	70	7	30	70	100	9M	4	2	62	68	10	8	80	0.7	M5	12	49	24 V
J.	20FXA	120	166	30	60	80	7	35	80	120	MIO	5	2.5	72	78	12	10	92	0.7	M6	8.3	69	1
6	40FXA	220	195	33.5	68	90	7	38.5	89	150	01M	5	3	82	88	14	10	110	0.7	M6	7.4	77	
٦	60FXA	300	210	35	73	100	8.5	38	100	150	01M	5	2	92	86	16	10	120	0.7	M6	6.6	87	
8	BOFXA	400	240	42	81	110.5	8.5	42	112	150	M12	5	3	102	108	16	10	140	0.7	M6	5.9	100	



	NOTE :
	ALSO
ALSO	NOTE : ALSO AVAILABLE WITH GEAR ADAPTER PLATES AND GEARED DRIVEN END (FXA Series) ALSO AVAILABLE 60FXA, 80FXA AND 120FXA ON REQUEST
AVAI	MTH
LABLE	GEAR
SPECIAL	ADAPTER ABLE 60F
ALSO AVAILABLE SPECIAL BACKLASH FREE VERSIONS ON REQUEST	E WITH GEAR ADAPTER PLATES AND GEARED DRIVEN END () ALSO AVAILABLE GOFXA, BOFXA AND 120FXA ON REQUEST
SH FR	AND
EE VERS	SEARED 120FXA
ONS O	ORIVEN
ON REO	EOUES
DUEST	(FXA
	Series)

SR	MODEL	TORQUE	OVE	RALL	OVERALL DIMENSIONS	SIONS			DIN	DRIVE END	(mm)		DIMEN	DRIVE END	(mm)		AR	COIL	CONSUMP.	VOLTAGE
NO.	NO.	IN (kg-m)	A	в	c	D	m	(mm)	٦	NO.OF	SIZE	Ρ	0	٦	R	s	~	at 25°C a (ohms)	at 25°C (WATTS)	(VOLTS
1	01FXB	10	82	23	37	2	8	34 × 36 × 6.3	41	3	M4	55	9	6	8	35	0.5	34	17	
2	02FXB	20	95	20	38	1.5	8	40 × 36 × 7.8	50	4	M6	65	9	6	8	45	0.7	24	24	
3	05FXB	. 35	114	23	43	1.5	10	50 x 46 x 9.8	60	4	M 6	80	9	6	8	53	0.7	14	42	24 v
4	10FXB	60	140	26	51	2	10	68 x 62 x 10.8	80	4	M6	100	12	10	10	70	0.7	12	49	
5	20FXB	120	166	30	60	2	10	78 × 72 × 12.10	92	5	M10	120	12	12	12	80	0.7	8.3	69	
6	40FXB	220	195	33.5	63	2	10	88 x 82 x 14.10	110	5	M10	150	15	15	15	89	0.7	7.4	77	

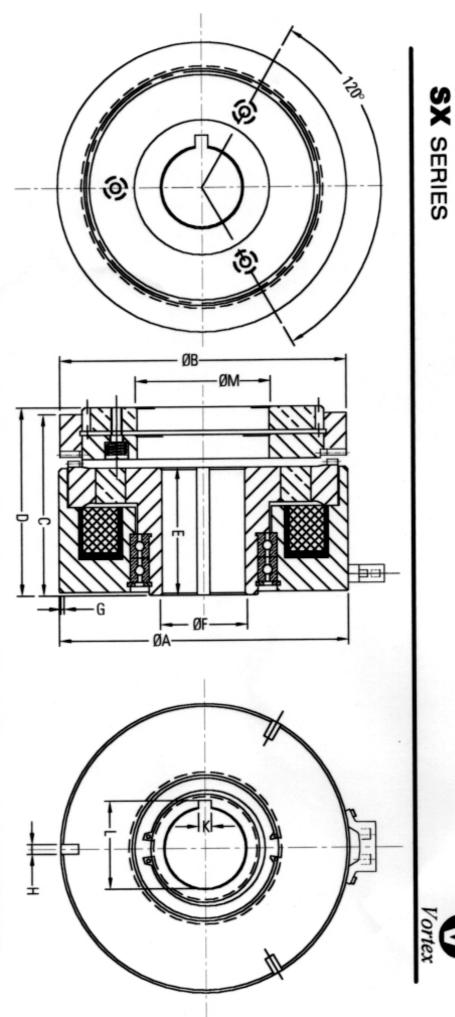


Vortex

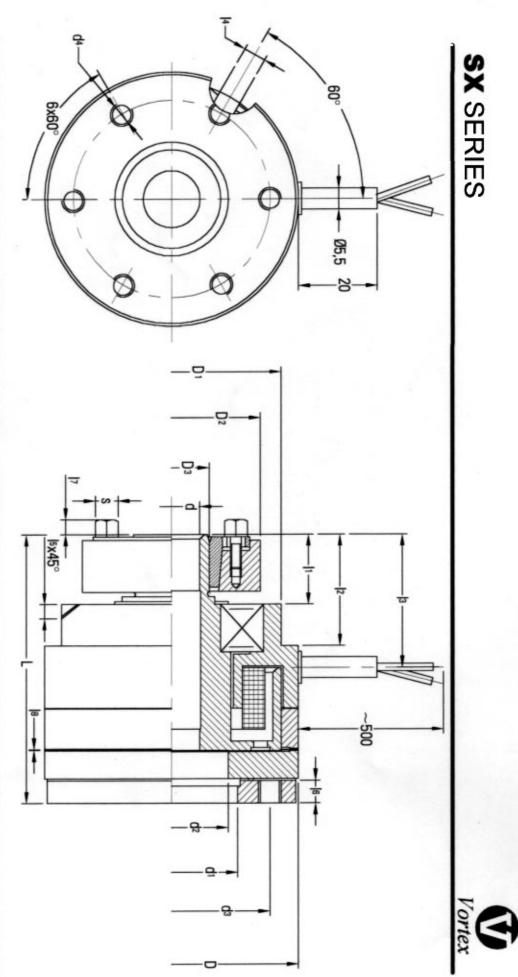
FX SERIES

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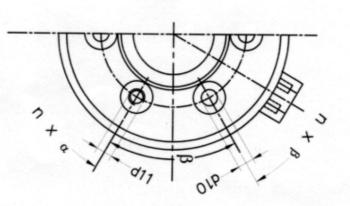
7	0	σı	4	з	N	_	NO	SR
60 SXA	40 SXA	20 SXA	10 SXA	05 SXA	02 SXA	01 SXA	NO.	
300	200	140	60	30	20	10	₹	TORQUE
210	195	166	134	114	95	82	>	OVE
210	195	162	127	109	93	80	в	RALL
111	96	90	80	66	63	54	0	OVERALL DIMENSIONS (mm)
113	99	93.5	83	69	62	57	D	NOISN
74	65	61	54	44	41	37	m	IS (m
105	100	79	62	56	46	38	Σ	3
68	65	60	46	38	35	25	п	MO
8	8	6	ŋ	4	4	з	G	MOUNTING DIMENSIONS
12	12	8	8	8	6	6	I	
20	18	18	14	10	10	8	*	MENS
72.1	68.1	63.1	48.6	40.1	37.1	26.7	-	SNO
0.4	0.4	0.7	0.7	0.7	0.7	0.5	(mm)	
ы	4	з	7	9	11	14	at 25°C (ohms)	RESISTANCE
170	140	115	85	64	52	42	at 25°C (WATTS)	.D
				24 v			(VOLIS)	VOLTAGE

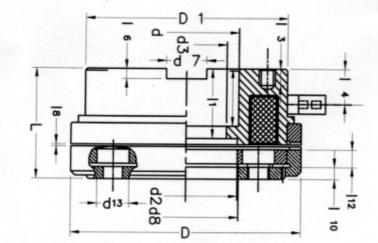


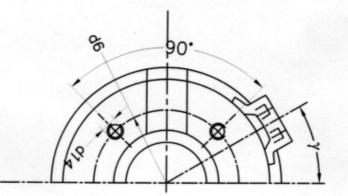
-		3 10SXB	2 05SXB	1 02SXB		SR. NO. Size
	3 150	3 75	3 50	25		Torque
1500	1500	1500	1500	1500	abaaa	Max.
57	36	27	21	19	Posse	Input
40	OE	25	20	15	VIIIII	bore
12	12	12	0	σι	screws	Torque for
10	10	10	8	8		8
134	114	96	82	67	o	
127	105	88	76	68	Ņ	
100	80	72	60	47	Ď	
55	44	36	30	20	ō	
68	62	52	47	36	à	
67	01 4	48	38	30	å	
105	90	75	65	52	0	
105 M12	MIO	M8	M	MB	å	Dimensions (mm)
123	104	83	84	71	-	
8	81	73	67	57	-	n) sions
36	30.8	28	26.5	18.5	H	
56	48.8	42	38.5	29.5	2	
63	54.8	48	44.5	35.2	Ŀ	
10	10	10	8	0	*	
8	8	0	6	O1	-	
13.5	11.5	9.5	7.5	6.0	•	
4	4	4	4	4	ŀ,	
0.2	0.2	0.2	0.2	0.2	-	A.G

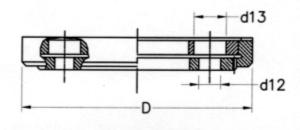


0	•	1 0		л.	4	•		•	-		NO.	
40RXA	ZURXA	IUKXA	AVNCO	DEDUA NA	DOBY A	OIRXA	U.SKXA	0.0000	0 SERVA		SIZE	
2200	0021	000	000	200	300	100	t	-	00		TORQUE	
82	6/	00	10	57	200	22	14.0	10.0	10 5	W	POWER	
80	0	8	42	5 0	1	25	20	3	1	٩		
195	166	134	14	5	2	82	67		2	0		
178	150	120	95	0.0	2	74	60	8	5	ō		
100	8	72	62	-	+	42	32	6	8	4		
80	65	60	45	3/		31	27	20	3	da		
150	100	8	75	8		50	46	đ	5	de		
95	80	68	55	4		36.5	31	22.5	2	dz		
11.5	9.5	9.5	7.8	5.5	+	4.5	4.5	1		oth		
10.5	8.5	8.5	6.8	6.8	0	5.8	4.8	4.0		d12		
M12	M12	M12	M 8	8	-	MA	MS	M4		di.		
68.5	63.5	57	50	4	01.0	34 5	31	27	1	-	DIME	
34	30	29	27	22	Г	10	19	17		-	DIMENSIONS	
12	10	8	8	J	0		U,	4	1	-	S (mm)	
20	20	16	14	12	-	13	10	10	1	4	5	
0.5	0.5	0.4	0.4	0.4	0.0	2	0.3	0.2	1	-		
28	25	22	22	20	11	17	14	14	-	-		
150	120	95	08	70	00	5	46	36	4	4		
MIO	M10	MB	BW	9M	CW		M4	M4	-	4		
17.5	14	14	12	12	10	5	8	8	1	4		
12.5	10	12.5	11.5	9	0		7	6.5	•	F		
6	6	5	5	2.5	2.2	2	2.5	2.5	10	-		
11.7	11.4	8,4	6.5	6	4.8		3.5	3	011	5		
6.5	5.5	4.5	3.5	S	2.3		1.5	1.4	111	F		
74	13.1	=	9	8.7	6.1		4.8	4.3	112			





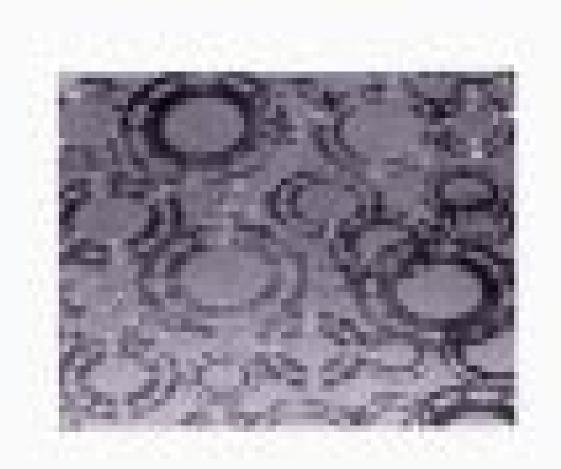




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